DESCRIPTION

FUEL INJECTION CONTROL APPARATUS FOR INTERNAL COMBUSTION ENGINE

5   Technical Field

The present invention relates to a fuel injection control apparatus for an internal combustion engine.

Background Arts

10   Conventionally, among direct injection internal combustion engines having a fuel injection valve (or an injector) for injecting fuel directly into the interior of the cylinder, one having a function of correcting the opening time of the fuel injection valve based on the

15   pressure in the interior of the cylinder (or the in-cylinder pressure) is known (see, for example, patent document 1 listed later).

The reason why this function is adopted is that the pressure in the interior of the cylinder (the in-cylinder

20   pressure) acts as a back pressure against the fuel injection valve. In this function, the in-cylinder pressure that changes in accordance with the running condition of the engine is calculated to correct the opening time of the fuel injection valve, thereby

25   obtaining a desired fuel injection quantity.

However, in the above-described conventional art, the in-cylinder pressure is calculated, and then the fuel

injection rate (i.e. the quantity of the fuel injected per unit time) is calculated based on the pressure difference between that in-cylinder pressure and the pressure of the fuel introduced to the fuel injection valve. Then, the

5   opening time of the fuel injection valve is calculated based on the calculated fuel injection rate and the required fuel quantity. Thus, in the conventional art as such, no consideration has been given to changes in the start time of the fuel injection.

10   This point will be discussed in the following with reference to Fig. 14. Fig. 14 shows changing behavior of the fuel injection rate. In Fig. 14, the vertical axis represents the fuel injection rate, and the horizontal axis represents the time. In Fig. 14, waveform X and

15   waveform Y of the fuel injection rate show changing behavior of the fuel injection rate for different in-cylinder pressures but the same rail pressure (i.e. the pressure of the fuel supplied to the fuel injection valve). Fuel injection rate waveform X represents the case in

20   which the in-cylinder pressure is a reference in-cylinder pressure serving as a reference (for example, the pressure under the condition in an injector characteristics measuring benchmark test (e.g. 1Mpa)), and fuel injection rate waveform Y represents the case with the engine in-

25   cylinder pressure in the internal combustion engine in a running state (e.g. 8Mpa).

     As will be seen from Fig. 14, when the in-cylinder

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pressure increases, the start time of the fuel injection
becomes earlier. If the start time of the fuel injection
becomes earlier, the fuel injection quantity will increase.

Fig. 15 is a fuel injection rate changing behavior
5   model serving as a model for the changing behavior of the
fuel injection rate shown in Fig. 14. The inventors of
the present invention modeled the changing behavior of the
fuel injection rate shown in Fig. 14 as a trapezoid shown
in Fig. 15. In Fig. 15, trapezoid X shown by a solid line
10   is a model for fuel injection rate waveform X in Fig. 14,
and trapezoid Y shown by a broken line is a model for fuel
injection rate waveform Y in Fig. 14.

In Fig. 15, letting Q be the area of trapezoid X or
the required fuel injection quantity, Qr be the area of
15   trapezoid Y or the actual injection quantity, dQ1 be the
variation in the fuel injection quantity due to the
variation in the fuel injection rate between in the case
of reference in-cylinder pressure and in the case of in-
cylinder pressure of the engine (i.e. the area within
20   trapezoid X above trapezoid Y in Fig. 15), and dQ2 be the
variation in the fuel injection quantity due to the
variation in the start time between in the case of
reference in-cylinder pressure and in the case of in-
cylinder pressure of the engine (i.e. the area within
25   trapezoid Y on the left side of trapezoid X in Fig. 15),
the actual injection quantity Qr can be represented by the
following formula (1)

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$$Qr = Q - dQ1 + dQ2 \qquad \text{---}(1)$$

Therefore, a command value for attaining desired fuel injection is represented by the following formula (2).

$$Q = Qr + dQ1 - dQ2 \qquad \text{---}(2)$$

In Fig. 15, letting A be the length of the upper base of trapezoid X, B be the length of the lower base of trapezoid X, Q' be the height of trapezoid X (i.e. the fuel injection rate in the case of reference in-cylinder pressure), q' be the height of trapezoid Y and C be the length of the portion of the upper base of trapezoid Y that overlaps trapezoid X, dQ1 is represented by the following formula (3)

$$dQ1 = Q - q$$
$$= Q - (B + C)q'/2$$
$$= Q - (B + (Aq'/Q' + B(Q' - q')/Q'))q'/2$$
$$= (1 - q'/Q')Q + (A - B)(q' - Q')q'/Q' \qquad \text{---}(3)$$

In the conventional art, dQ2 in formula (2) is not taken into consideration, and corrected fuel injection rate is obtained by the relationship represented by the following formula (4).

$$Qr = Qq'/Q' \qquad \text{---}(4)$$

Thus, in the conventional art, dQ1 is represented by the following formula (5).

$$dQ1 = Q - Qr$$
$$= (1 - q'/Q')Q \qquad \text{---}(5)$$

Equation (5) lacks the second term in equation (3), and therefore, no consideration has been given to an error

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corresponding to this term in the conventional art.

In addition, when the rail pressure is low, the change in the fuel injection rate after the start of the fuel injection is moderate. Therefore, the gradient of the left edge of the trapezoid is small and the value (A-B) is large. Thus, in the case that the rail pressure is low, influence of the value (A-B) is significant, and therefore, a large error will be introduced if dQ1 is obtained from equation (5).

[Patent Document 1] Japanese Patent Application Laid-Open No. 9-256886

[Patent Document 2] Japanese Patent Application Laid-Open No. 2000-54889

Disclosure of Invention

The present invention has been made in view of the above-described situation. An object of the present invention is to provide a technology for controlling the fuel injection quantity that changes with changes in the in-cylinder pressure with improved accuracy.

In order to attain the above object, according to the present invention, there is provided a fuel injection control apparatus for an internal combustion engine equipped with a fuel injection valve for directly injecting high pressure fuel supplied by high pressure fuel supply means into a cylinder, comprising:

in-cylinder pressure detection means for detecting

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or estimating in-cylinder pressure of said cylinder;

time period calculation means for calculating fuel injection time period over which fuel is injected from said fuel injection valve, the fuel injection time period being corrected to compensate a variation in the fuel injection quantity caused by a variation in the fuel injection rate due to a variation in the in-cylinder pressure detected or estimated by said in-cylinder pressure detection means relative to a reference in-cylinder pressure that is stored in advance;

start time variation calculation means for calculating a variation in the fuel injection start time at the in-cylinder pressure detected or estimated by said in-cylinder pressure detection means relative to the fuel injection start time at said reference in-cylinder pressure;

control means for controlling the time period over which fuel is injected from said fuel injection valve based on the fuel injection time period calculated by said time period calculation means and the variation in the fuel injection start time calculated by said start time variation calculation means.

By calculating a fuel injection time period while correcting a variation in the fuel injection quantity caused by a variation in the fuel injection rate and calculating a variation in the fuel injection start time, it is possible to control the fuel injection quantity that

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changes with changes in the in-cylinder pressure with
improved accuracy, and it is possible to attain a target
fuel injection quantity irrespectively of the running
state of the internal combustion engine.

5   The above-described apparatus may further include:

fuel pressure detection means for detecting the
pressure of the high pressure fuel supplied to said fuel
injection valve by said high pressure fuel supply means;

fuel injection quantity characteristic storing means
10   for storing a characteristic, in relation to valve opening
time of said fuel injection valve, of the fuel injection
quantity injected by said fuel injection valve during the
valve opening time in accordance with the pressure of the
high pressure fuel supplied to said fuel injection valve
15   by said high pressure fuel supply means;

required fuel injection quantity calculation means
for calculating a desired fuel injection quantity based on
the running state of the internal combustion engine;

fuel injection rate calculation means for
20   calculating fuel injection rate based on the fuel pressure
detected by said fuel pressure detection means and the in-
cylinder pressure detected or estimated by said in-
cylinder pressure detection means;

fuel injection quantity variation calculation means
25   for calculating a variation in the fuel injection quantity
caused by a variation in a second fuel injection rate
calculated by said fuel injection rate calculation means

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based on the in-cylinder pressure detected or estimated by
said in-cylinder pressure detection means relative to a
first fuel injection rate calculated by said fuel
injection rate calculation means based on said reference
5   in-cylinder pressure;

        coefficient calculation means for calculating a
variation in fuel injection delay time per unit in-
cylinder pressure for the fuel pressure detected by said
fuel pressure detection means.

10   In this case, said time period calculation means may
calculate the fuel injection time period utilizing said
fuel injection quantity characteristic storing means based
on the variation in the fuel injection quantity calculated
by said fuel injection quantity variation calculation
15   means and the fuel injection quantity calculated by said
required fuel injection quantity calculation means, and
said start time variation calculation means may calculate
the variation in the fuel injection start time based on a
variation in the in-cylinder pressure detected or
20   estimated by said in-cylinder pressure detection means
relative to said reference in-cylinder pressure and the
variation calculated by said coefficient calculation means.

        With the above features, it is possible to calculate
a correction value based on the reference in-cylinder
25   pressure. Therefore, a characteristic stored in the fuel
injection quantity characteristic storing means, for
example, a characteristic obtained by an injector

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characteristics measuring benchmark test, can be applied to an actual internal combustion engine directly.

The above-described apparatus may further includes:

a needle valve provided in said fuel injection valve
5   that moves in the axial direction to effect valve opening and closing operations;

fuel injection quantity estimation means for estimating, when fuel injection by said fuel injection valve is started, the quantity of fuel injected since the
10   valve opening operation of said needle valve is started until said needle valve reaches a full open state, based on the fuel pressure detected by said fuel pressure detection means and the in-cylinder pressure detected or estimated by said in-cylinder pressure detection means;

15   comparison means for comparing the estimated fuel quantity estimated by said fuel injection quantity estimation means and the fuel injection quantity calculated by said required fuel injection quantity calculation means,

20   wherein, said fuel injection quantity variation calculation means may calculate the variation in the fuel injection quantity using different calculation processes, in accordance with a result of the comparison by said comparison means, between in the case that said fuel
25   injection quantity is less than said estimated fuel quantity and in the case that said fuel injection quantity is more than or equal to said estimated fuel quantity.

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As per the above, by using different processes of calculating a variation in the fuel injection quantity depending on whether the needle valve has reached the full open state or not, it is possible to calculate the

5   variation in the fuel injection quantity by a simplified calculation method. Therefore, in the case that a relationship necessary for calculation of the variation in the fuel injection quantity is stored as a map, the data amount of the map can be made minimum.

10   In the above described apparatus, said fuel injection quantity variation calculation means may calculate the variation in the fuel injection quantity by modeling a change with time in the fuel injection rate as a polygon in a coordinate system and calculating a change

15   in the area of said polygon.

With this feature, it is possible to calculate the variation in the fuel injection quantity more simply.

Furthermore, in the above-described apparatus, said fuel injection quantity variation calculation means may

20   include suction chamber pressure calculation means for calculating pressure in a suction chamber formed in the tip end side of a valve seat on/from which said needle valve is to be received/detached, based on the fuel pressure detected by said fuel pressure detection means

25   and the position of said needle valve;

unit fuel injection quantity variation calculation means for calculating a variation in the fuel injection

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quantity per unit in-cylinder pressure based on the fuel
injection quantity calculated by said required fuel
injection quantity calculation means and the suction
chamber pressure calculated by said suction chamber
5   pressure calculation means,

wherein, when according to a result of the
comparison by said comparison means, said fuel injection
quantity is less than said estimated fuel injection
quantity, said fuel injection quantity variation
10   calculation means may calculate the variation in the fuel
injection quantity based on a variation in the in-cylinder
pressure detected or estimated by said in-cylinder
pressure detection means relative to said reference in-
cylinder pressure and the variation in the fuel injection
15   quantity per unit in-cylinder pressure calculated by said
unit fuel injection quantity variation calculation means.

With the above features, when the needle valve has
not reached the full open state, the variation in the fuel
injection quantity can be calculated based on the pressure
20   of the suction chamber. Therefore, correction with
improved accuracy can be made possible.

Still further, the above-described apparatus may
further include:

fuel pressure detection means for detecting the
25   pressure of the high pressure fuel supplied to said fuel
injection valve by said high pressure fuel supply means;

fuel injection quantity characteristic storing means

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for storing a characteristic, in relation to valve opening
time of said fuel injection valve, of the fuel injection
quantity injected by said fuel injection valve during the
valve opening time in accordance with the pressure of the
5   high pressure fuel supplied to said fuel injection valve
by said high pressure fuel supply means;

required fuel injection quantity calculation means
for calculating a desired fuel injection quantity based on
the running state of the internal combustion engine;

10   first virtual fuel pressure calculation means for
calculating a first virtual fuel pressure by subtracting a
variation in the in-cylinder pressure detected or
estimated by said in-cylinder pressure detection means
relative to said reference in-cylinder pressure from the
15   fuel pressure detected by said fuel pressure detection
means;

second virtual fuel pressure calculation means for
calculating a second virtual fuel pressure by adding a
variation in the in-cylinder pressure detected or
20   estimated by said in-cylinder pressure detection means
relative to said reference in-cylinder pressure to the
fuel pressure detected by said fuel pressure detection
means;

injection delay time calculation means for
25   calculating injection delay time from the time at which a
signal for opening said fuel injection valve is generated
to the time at which fuel injection by said fuel injection

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valve is started, based on the fuel pressure detected by
said fuel pressure detection means,

wherein, said time period calculation means may
calculate the fuel injection time period utilizing said
fuel injection characteristic storing means based on the
first virtual fuel pressure calculated by said first
virtual fuel pressure calculation means and the fuel
injection quantity calculated by said required fuel
injection quantity calculation means, and

said start time variation calculation means may
calculate, by means of said fuel injection delay time
calculation means, a fuel injection delay time for the
fuel pressure detected by said fuel pressure detection
means and a fuel injection delay time for the second
virtual fuel pressure calculated by said second virtual
fuel pressure calculation means and calculates the
variation in the fuel injection start time from a
difference between those injection delay times.

By introducing a virtual fuel pressure as per the
above, it is possible to calculate the time period over
which the fuel is injected in a more simple way.

Brief Description of Drawings

Fig. 1 schematically shows the basic structure of a
diesel engine as an internal combustion engine in which a
fuel injection control apparatus according to a first
embodiment of the present invention is applied.

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Fig. 2 is a cross sectional view schematically showing a fuel injection valve in the first embodiment of the present invention.

Fig. 3 is a block diagram showing the fuel injection

5   control apparatus according to the first embodiment of the present invention.

Fig. 4 shows, in Part (A), a fuel injection rate changing behavior model in which the changing behavior of the fuel injection rate is modeled as a trapezoid, and

10   shows, in Part (B), a state in which the trapezoid shown in Part (A) is divided into two portions.

Fig. 5 is a diagram for illustrating a method of calculating a variation in the fuel injection quantity in the case that the fuel injection was completed before the

15   needle valve reached the fully lifted state.

Fig. 6 is a diagram for illustrating correction of the variation in the fuel injection start time; Part (A) of Fig. 6 shows a relationship between the driving signal and the fuel injection rate before the correction, and

20   Part (B) of Fig. 6 shows a relationship between the driving signal and the fuel injection rate after the correction.

Fig. 7 shows the relationship between the proportionality coefficient (or the sensitivity) α and the

25   rail pressure Pcr in the first embodiment of the present invention.

Fig. 8 shows the relationship between variations in

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the fuel injection delay time $\Delta\tau d$ and variations in the in-cylinder pressure ΔPcl.

Fig. 9 is a flow chart of a process of calculating correction value of the fuel injection quantity in the first embodiment of the present invention.

Fig. 10 is a block diagram showing fuel injection quantity variation calculation means and related portions in the second embodiment of the present invention.

Fig. 11 is a block diagram showing a fuel injection control apparatus according to a third embodiment of the present invention.

Fig. 12 is a τ-Q map in the third embodiment of the present invention.

Fig. 13 shows the relationship between the rail pressure Pcr and the fuel injection delay time τd.

Fig. 14 is a graph showing a changing behavior of fuel injection rate.

Fig. 15 shows a fuel injection rate changing behavior model serving as a model for the changing behavior of the fuel injection rate.

Best Mode For Carrying Out The Invention

In the following, the best mode for carrying out the present invention will be described in detail based on exemplary embodiments.

<Embodiment 1>

Fig. 1 schematically shows the basic structure of a

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diesel engine as an internal combustion engine in which a
fuel injection control apparatus according to a first
embodiment of the present invention is applied.

As shown in Fig. 1, the internal combustion engine 1
is mainly composed of a fuel supply system 2, cylinders
(or combustion chambers) 3, intake passage 4 and exhaust
passage 5. The internal combustion engine 1 is an in-line
four cylinder diesel engine in which four cycles including
intake stroke, compression stroke, explosion stroke (or
expansion stroke) and exhaust stroke are repeated to
create an output.

The fuel supply system 2 includes a supply pump 6, a
common rail 7, fuel injection valves 8, fuel passage 9
etc.. The supply pump 6 raises the pressure of the fuel
drawn from a fuel tank (not shown) to a high pressure and
supplies the fuel to the common rail 7 through the fuel
passage 9. The common rail 7 functions as a pressure
accumulation chamber for maintaining (or accumulating) the
pressure of high pressure fuel supplied from the supply
pump 6 at a predetermined pressure and distributes the
pressurized fuel to the respective fuel injection valves 8
through the fuel pipes connected to the common rail 7.
The high pressure fuel supply means comprises the supply
pump 6, the common rail 7 and the fuel passage 9.

The fuel injection valve 8 is an electromagnetically
driven on-off valve equipped with a electromagnetic
solenoid (not shown) in the interior thereof. The fuel

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injection valve 8 is opened fitly to supply an appropriate
quantity of the fuel pressurized in the common rail 7 into
the interior of the cylinder 3 by direct injection at an
appropriate timing. The fuel injection valve 8 may be

5   equipped with a piezoelectric actuator instead of the
electromagnetic solenoid.

        The internal combustion engine 1 is provided with
various sensors such as an accelerator position sensor
that outputs a signal indicative of the travel of the

10   accelerator pedal (not shown) resulting from depression by
the driver, a crank position sensor that outputs a signal
indicative of the number of engine revolutions of the
crankshaft (not shown), a water temperature sensor that
outputs a signal indicative of the temperature of cooling

15   water circulating in the internal combustion engine 1 (the
cooling water temperature), an air flow meter outputting a
signal indicative of the flow rate of the air introduced
into the cylinder 3 through the intake passage 4 (the
intake air quantity) and a rail pressure sensor 11 that

20   detects the pressure of the high pressure fuel under
pressure in the common rail 7 (the rail pressure). The
signals of these sensors are input to an electronic
control unit (ECU) 10. The rail pressure sensor 11
constitutes the fuel pressure detection means.

25   ECU 10 includes a logical operating circuit composed
of a central processing unit (CPU), a read only memory
(ROM), a random access memory (RAM), a backup RAM and

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other elements. The ECU 10 performs overall control of
various components of the internal combustion engine 1
based on signals from various sensors. For example, the
ECU 10 detects the running state of the internal

5   combustion engine 1 to control the opening/closing
operation of the fuel injection valves 8.

Furthermore, the ECU 10 executes inputting of
signals output from various sensors, calculation of the
number of engine revolutions, calculation of the load,

10   calculation of the fuel injection quantity etc. in a basic
routine that is to be performed at regular intervals.
Various signals input into the ECU 10 and various control
values obtained through the calculation by the ECU 10 in
the basic routine are temporarily stored in the RAM of the

15   ECU 10. Still further, in an interrupting process that is
triggered by, for example, signal inputs from various
sensors or switches, lapse of a certain time or input of a
pulse signal from a crank position sensor, the ECU 10 .
reads out various control values from the RAM and executes

20   the fuel injection control or other control in accordance
with those control values. The ECU 10 constitutes the
fuel injection control apparatus, and constitutes the time
period calculation means 51, the start time variation
calculation means 52, the control means 53, the fuel

25   injection quantity characteristic storing means 54, the
required fuel injection quantity calculation means 55, the
fuel injection rate calculation means 56, the fuel

injection quantity variation calculation means 57, the coefficient calculation means 58, the fuel injection quantity estimation means 59 and comparison means 60 in Fig.3 described hereinafter.

5 In the following, the fuel injection valve 8 will be more specifically described. Fig. 2 is a cross sectional view schematically showing the fuel injection valve 8 in this embodiment.

As shown in Fig. 2, the fuel injection valve 8

10 includes a main body 22 having a fuel injection hole 21 at its tip end, a needle-shaped needle valve 23 (i.e. the valve body) and a coil spring 24 that biases the needle valve 23 in the closing direction. The needle valve 23 is disposed in the interior of the main body 22 in such a way

15 as to be movable in the axial direction. The needle valve 23 closes the fuel injection hole 21 when received on a valve seat 25 in its advanced state and opens the fuel injection valve 21 when detached from the valve seat 25 in its retracted state. The fuel injection hole 21 is

20 provided at a suction chamber 26 provided in the tip end side of the valve seat 25 in the main body 22.

Furthermore, the fuel injection valve 8 includes a first fuel supply passage 31 for introducing high pressure fuel supplied from the common rail 7 with a predetermined

25 pressure to the fuel injection hole 21, a control chamber 32 for receiving high pressure fuel to press the needle valve 23 in the closing direction, a second fuel supply

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passage 33 branching from the first fuel supply passage 31

for introducing high pressure fuel supplied from the

common rail 7 with a predetermined pressure to a control

chamber 32 and a fuel discharge passage 34 for discharging

5   the high pressure fuel in the control chamber 32 to reduce

the fluid pressure in the control chamber 32.

In the second fuel supply passage 33, an inlet

orifice 33a that determines the flow rate of the fuel

flowing into the control chamber 32 is provided. In the

10   fuel discharge passage 34, an outlet orifice 34a that

determines the fuel discharge amount is provided. The

ratio of the cross sectional area of the inlet orifice 33a

and the outlet orifice 34a is designed in such a way that

the cross sectional area of the outlet orifice 34a is

15   larger than that of the inlet orifice 33a. For example,

the ratio is 2:3.

The needle valve 23 has a main piston 23a that faces

the control chamber 32 and receives the fuel pressure in

the control chamber 32 to move the needle valve 23

20   downwardly. A sub piston 23c is provided in the fuel

injection hole 21 side of the needle valve 23 relative to

the main piston 23a. At a position in the first fuel

supply passage 31 leading to the fuel injection hole 21,

there is provided a fuel reservoir 31a in such a way as to

25   face the sub piston 23c. Thus, the pressure of the fuel

in the fuel reservoir 31a acts on the sub piston 23c to

press the needle valve 23 in the opening direction (i.e.

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the upward direction in Fig. 2). The area Ss over which
the sub piston 23c receives the pressure of the fuel in
the fuel reservoir 31a is designed to be smaller than the
area Sm over which the main piston 23a receives the
5   pressure of the fuel in the control chamber 32. In
addition, a coil spring 24 for biasing the needle valve 23
in the closing direction is provided in the main piston
23a side of the sub piston 23c.

Letting Fm be the pressing force exerted on the main
10   piston 23a by the pressure of the fuel in the control
chamber 32, Fs be the pressing force exerted on the sub
piston 23c by the pressure of the fuel in the fuel
reservoir 31a, Fc be the biasing force of the coil spring
24, inequalities Fm+Fc>Fs and Fc<Fs hold in the steady
15   state.

Furthermore, a back pressure control valve 35 for
sealing, in its closed state, high pressure fuel in the
control chamber 32 and for letting, in its opened state,
fuel out of the control chamber 32 to the fuel discharge
20   passage 34 is provided intervening in the fuel discharge
passage 34 from the control chamber 32. The back pressure
control valve 35 is composed of an electromagnetic valve
and provide in the interior of the main body 22. When the
back pressure control valve 35 is in the closed state, the
25   pressure of the fuel in the control chamber 32 increases
to press the main piston 23a to move the needle valve 23
downwardly in cooperation with the biasing force of the

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coil spring 24.

In this process, although fuel having the pressure
same as the pressure in the control chamber 32 is
introduced from the first fuel supply passage 31 into the
fuel reservoir 31a to press the sub piston 23c, its
pressing force Fs cannot match the cooperative force Fm+Fc.
Consequently, the needle valve 23 is maintained in the
state in which it closes the fuel injection hole 21.

After that, when the back pressure control valve 35
is opened, fuel is discharged from the control chamber 32
through the fuel discharge passage 34. In this process,
since the outlet orifice 34a is designed to be larger than
the inlet orifice 33a, the quantity of the fuel flowing
out of the control chamber 32 is more than the quantity of
the fuel flowing into the control chamber 32.
Consequently, the fuel pressure in the control chamber 32
falls.

Then at the time when Fm+Fc<Fs is established, the
needle valve 23 lifts (i.e. moves upward, and opens the
valve), so that the fuel injection hole 21 is opened and
the fuel injection is started.

Here, a fuel injection valve drive control process
for driving the fuel injection valve 8 will be described.
The fuel injection valve drive control process is executed
by the ECU 10.

Before fuel injection, the ECU 10 keeps the back
pressure control valve 35 in the closed state, and the

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interior of the control chamber 32 is filled with high

pressure fuel introduced from the common rail 7 through

the second fuel supply passage 33. Thus, the piston 23a

of the needle valve 23 is in the lowered position and the

5   fuel injection hole 21 is closed.

When the time for fuel injection comes, the back

pressure control valve 35 is opened by a command from the

ECU 10, and high pressure fuel in the interior of the

control chamber 32 is discharged through the fuel

10   discharge passage 34. Consequently, the fuel pressure in

the control chamber 32 falls, so that the needle valve 23

lifts to open the fuel injection hole 21.

When a predetermined fuel injection time elapses

after that, the back pressure valve 35 is closed by the

15   ECU 10. Then, high pressure fuel flows into the control

chamber 32 and is sealed therein. Thus, the pressure in

the control chamber 32 rises, so that the needle valve 23

goes down to close the fuel injection hole 21.

Fig. 3 is a block diagram of the fuel injection

20   control apparatus according to this embodiment. As shown

in Fig.3, the fuel injection control apparatus according

to this embodiment is equipped with a time period

calculation means 51, start time variation calculation

means 52 and control means 53.

25   The time period calculation means 51 calculates the

fuel injection time period being corrected to compensate a

variation in the fuel injection quantity caused by a

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variation in the fuel injection rate due to a variation in
the in-cylinder pressure detected or estimated by the in-
cylinder pressure detection means 61 relative to a
reference in-cylinder pressure that is stored in advance.

5   The start time variation calculation means 52
calculates a variation in the fuel injection start time at
the in-cylinder pressure detected or estimated by in-
cylinder pressure detection means 61 relative to the fuel
injection start time at the reference in-cylinder pressure

10   described later.

The control means 53 controls the time period over
which fuel is injected from the fuel injection valve 8
based on the fuel injection time period calculated by the
time period calculation means 51 and the variation in the

15   fuel injection start time calculated by the start time
variation calculation means 52.

The fuel injection control apparatus according to
this embodiment is further equipped with the fuel
injection quantity characteristic storing means 54 for

20   storing a characteristic, in relation to valve opening
time of the fuel injection valve 8, of the fuel injection
quantity injected by the fuel injection valve 8 during the
valve opening time in accordance with the pressure of the
high pressure fuel supplied to the fuel injection valve 8;

25   the required fuel injection quantity calculation means 55
for calculating a desired fuel injection quantity based on
the running state of the internal combustion engine; the

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fuel injection rate calculation means 56 for calculating
fuel injection rate based on the fuel pressure detected by
the rail pressure sensor 11 and the in-cylinder pressure
detected or estimated by the in-cylinder pressure

5   detection means 61; the fuel injection quantity variation
calculation means 57 for calculating a variation in the
fuel injection quantity caused by a variation in a second
fuel injection rate calculated by the fuel injection rate
calculation means 56 based on the in-cylinder pressure

10   detected or estimated by the in-cylinder pressure
detection means 61 relative to a first fuel injection rate
calculated by the fuel injection rate calculation means 56
based on the reference in-cylinder pressure; the
coefficient calculation means 58 for calculating a

15   variation in fuel injection delay time per unit in-
cylinder pressure for the fuel pressure detected by the
rail pressure sensor 11.

The time period calculation means 51 calculates the
fuel injection time period utilizing the fuel injection

20   quantity characteristic storing means 54 based on the
variation in the fuel injection quantity calculated by the
fuel injection quantity variation calculation means 57 and
the fuel injection quantity calculated by the required
fuel injection quantity calculation means55.

25   The start time variation calculation means 52
calculates the variation in the fuel injection start time
based on a variation in the in-cylinder pressure detected

or estimated by the in-cylinder pressure detection means
61 relative to the reference in-cylinder pressure and the
variation calculated by the coefficient calculation means
58.

5   Furthermore, the fuel injection quantity estimation
means 59 estimates the quantity of fuel injected since the
valve opening operation of the needle valve 23 is started
until the needle valve 23 reaches a full open state, based
on the fuel pressure detected by the rail pressure sensor
10   11 and the in-cylinder pressure detected or estimated by
the in-cylinder pressure detection means 61.

The comparison means 60 compares the estimated fuel
quantity estimated by the fuel injection quantity
estimation means 59 and the fuel injection quantity
15   calculated by the required fuel injection quantity
calculation means 55.

The fuel injection quantity variation calculation
means 57 calculates the variation in the fuel injection
quantity using different calculation processes, in
20   accordance with a result of the comparison by the
comparison means 60, between in the case that said fuel
injection quantity is less than the estimated fuel
quantity and in the case that said fuel injection quantity
is more than or equal to the estimated fuel quantity.

25   In the following, a method of correcting the fuel
injection quantity will be described in particular.

In the case that fuel is directly injected into the

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cylinder 3 by the fuel injection valve 8, the in-cylinder
pressure serving as the back pressure changes in
accordance with the running state of the engine.
Therefore, even if the ECU 10 commands to inject a
5   predetermined quantity of fuel, variations are generated
in the actual fuel injection quantity.

The internal combustion engine 1 is a direct
injection internal combustion engine equipped with the
fuel injection valve 8 for injecting fuel into the
10   cylinder directly, and so the fuel injection hole 21 is
disposed in the interior of the cylinder 3. When the fuel
injection hole 21 is opened, as described above, the back
pressure control valve 35 is opened by a command from the
ECU 10 and high pressure fuel in the interior of the
15   control chamber 32 is discharged through the fuel
discharge passage 34. Thus, the fuel pressure in the
control chamber 32 falls, so that the needle valve 23 will
lift. In this process, since the fuel injection hole 21
is disposed in the interior of the cylinder 3, the in-
20   cylinder pressure is exerted on the needle valve 23.
Since the in-cylinder pressure acts on the needle valve 23
in the lifting direction, the opening timing of the fuel
injection hole 21 is advanced, namely the fuel injection
is started at an earlier time.

25   In view of this, in this embodiment, a variation in
the fuel injection quantity caused by a variation in the
fuel injection rate at the in-cylinder pressure of the

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engine (a detected or estimated pressure in the running
state of the internal combustion engine) relative to the
fuel injection rate at a reference in-cylinder pressure
(for example, the condition in an injector characteristics

5   measuring benchmark test (e.g. 1Mpa)) is calculated (or
estimated) and, in addition, a variation in the fuel
injection start time is corrected to control the power
supply time during which electric power is supplied to the
fuel injection valve 8 (i.e. to the back pressure control

10   valve 3) (namely, the time period over which fuel is
injected by the fuel injection valve 8) is controlled. In
other words, the time period over which fuel is injected
from the fuel injection valve 8 is controlled by the
control means 53 based on the fuel injection time period

15   calculated by the time period calculation means 51 and the
variation in the fuel injection start time calculated by
the start time variation calculation means 52.

        Firstly, a method for calculating a variation in the
fuel injection quantity due to a variation in the fuel

20   injection rate will be described.

        In this embodiment, a fuel injection rate changing
behavior model in which the changing behavior of the fuel
injection rate is modeled in a similar manner as Fig. 15
that was referred to in the description of background arts

25   is used (see Fig. 4), and correction of the fuel injection
quantity is performed utilizing the fuel injection rate
changing behavior model.

In addition, in this embodiment, the calculation process is switched depending on whether the lifting distance (moving distance) of the needle valve 23 reaches one of the fully lifted states(the position at which the needle valve 23 lifts (opens) completely or is in the fully opened state) or not, Wherein, the needle valve 23 lifts in accordance with the desired fuel injection quantity (requested fuel injection quantity) based on the running condition of the engine.

In other words, the calculation process is switched between in the case in which the fuel injection was effected after or at the same time when the needle valve 23 had reached the fully lifted state and in the case in which the execution of the fuel injection was completed before the needle valve 23 had reached the fully lifted state. In connection with this, determination as to whether the needle valve had reached the fully lifted state is made by comparing a fuel injection quantity calculated by the required fuel injection quantity calculation means 55 and an estimated fuel quantity calculated by the fuel injection quantity estimation means 59 (a fuel quantity supposed to be injected until the needle valve 23 reaches the fully lifted state, which is estimated based on the fuel pressure detected by the rail pressure sensor 11 and the in-cylinder pressure detected or estimated by the in-cylinder pressure detection means 61 (which will be described later)) by the comparison

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means 60. Based on the result of the comparison by the
comparison means 60, a variation in the fuel injection
quantity is calculated using calculation process that is
different between in the case in which the fuel injection
5   quantity is less than or equal to the estimated fuel
quantity and in the case in which the fuel injection
quantity is more than the estimated fuel quantity. In the
following both of the above cases will be described.

        First, a description will be made on the case in
10   which the fuel injection was effected after or at the same
time when the needle valve 23 had reached the fully lifted
state.

        Part (A) in Fig. 4 shows a fuel injection rate
changing behavior model in which the changing behavior of
15   the fuel injection rate is modeled as a trapezoid. Part
(B) of Fig. 4 shows a state in which the trapezoid shown
in Part (A) in Fig. 4 is divided into two portions.

        In Fig. 4, trapezoid X shown by the solid line is a
model for the fuel injection rate waveform X (in the case
20   of reference in-cylinder pressure) shown in Fig. 14, as
with Fig. 15, and trapezoid Y shown by the broken line is
a model for the fuel injection rate waveform Y (in the
case of the engine in-cylinder pressure) shown in Fig. 14.

        Position (coordinates) a shown in Figs. 4(A) and
25   4(B) is set to represent the fuel injection rate at the
time when the needle valve 23 reaches the fully lifted
state. An additional line is drawn from this position a

parallel to the right edge of trapezoid X, so that the trapezoid X is divided into a triangle X1 and a parallelogram X2 as shown in Part (B) in Fig. 4

Here, as shown in Figs. 4(A) and 4(B), the vertexes of trapezoid X are designated by a, b, c and d, and the vertexes of trapezoid Y are designated by e, f, g and h. In addition the intersection point of the left edge ab of trapezoid X and the upper base eh of trapezoid Y is designated by i, the intersection point of the additional line drawn from vertex a parallel to the right edge dc and the lower base bc is designated by j, and the intersection point of the left edge aj of parallelogram X2 and the upper base eh of trapezoid Y is designated by k.

The variation ΔQ in the fuel injection quantity with a change in the fuel injection rate is calculated by the following equation (6). Here, the variation ΔQ of the fuel injection quantity with a change in the fuel injection rate corresponds to the area of trapezoid aihd. The variation ΔQ of the fuel injection quantity can be represented as the sum of the variation Δq1 until the needle valve 23 reaches the fully lifted state (the area of triangle aik) and the variation Δq2 after the needle valve has reached the fully lifted state (the area of parallelogram akhd).

Namely, the variation ΔQ in the fuel injection quantity is represented as follows.

$$ΔQ = Δq1 + Δq2 \qquad --(6)$$

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Here, the fuel injection quantity at the time when the needle valve 23 reached the fully lifted state, or the area of triangle abj is represented by Qfl, the required fuel injection quantity, or the area of trapezoid X is represented by Q, the fuel injection rate at the reference in-cylinder pressure, or the height of trapezoid X is represented by Q', and the fuel injection rate at the in-cylinder injection, or the height of trapezoid Y is represented by q'. Then, the height of triangle aik and parallelogram akhd is represented by $(Q'-q')$ and the area of parallelogram ajcd is represented by $(Q-Qfl)$. Therefore, the variations $\Delta q1$ and $\Delta q2$ can be calculated by the following formulas (7) and (8) based on the ratio of the areas.

$$\Delta q1 = Qfl \times (1 - q'/Q')^2 \qquad -- (7)$$

$$\Delta q2 = (Q - Qfl) \times (1 - q'/Q') \quad -- (8)$$

Here, the fuel injection rate can be interpreted as an orifice flow, and the fuel injection rates Q' and q' can be calculated by the following formulas (9) and (10) respectively. In connection with this, the orifice coefficient is represented by C0, the injection hole area (i.e. the area of the aperture of the fuel injection valve) is represented by A, the rail pressure is represented by Pcr, the in-cylinder pressure at the time of starting the fuel injection is represented by Pcl, the reference in-cylinder pressure is represented by P0 and the density of the fuel is represented by ρ.

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$$Q' = C0 \times A \times (2 \times (Pcr - P0)/ \rho)^{1/2} \qquad --(9)$$

$$q' = C0 \times A \times (2 \times (Pcr - Pcl)/ \rho)^{1/2} \qquad --(10)$$

Therefore, the variation ratio (q'/Q') of the fuel injection rate can be represented by the following formula (11).

$$q'/Q' = ((Pcr - Pcl)/(Pcr - P0))^{1/2} \quad --(11)$$

In the case that the fuel injection is effected after or at the same time when the needle valve 23 has reached the fully lifted state, a variation ΔQ in the fuel injection quantity caused by a change in the fuel injection rate can be calculated by the above-described method.

Next, a description will be made of the case in which the fuel injection was completed before the needle valve 23 reached the fully lifted state.

Fig. 5 is a diagram for illustrating a method of calculating a variation in the fuel injection quantity in the case that the fuel injection was completed before the needle valve 23 reached the fully lifted state.

In Fig. 5, triangle X1 is a model for the waveform of the fuel injection rate at the time when the needle valve 23 reached the fully lifted state as with triangle X shown in Part (B) in Fig. 4. The hatched triangle X1' is a model for the waveform of the fuel injection rate in the case that the fuel injection was completed before the needle valve 23 had reached the fully lifted state.

The variation $\Delta Q1$ in the fuel injection quantity in

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the case that the fuel injection was completed before the
needle valve 23 reached the fully lifted state can be
obtained based on the ratio of the area of triangle X1 and
the area of triangle X1'.

5   The areas of triangle X1 is represented by Qf1, and
the area of triangle X1' corresponds to the fuel injection
quantity Q as described above. Thus, the ratio of the
areas is represented by the following formula (12).

$$\Delta Q1/\Delta q1 = Q/Qf1 \qquad --(12)$$

10   Therefore, the variation $\Delta Q1$ in the fuel injection
quantity in the case that the fuel injection was completed
before the needle valve 23 reached the fully lifted state
can be represented by the following formula (13).

$$\Delta Q1 = Q \times (1 - q'/Q')^2 \qquad --(13)$$

15   Next, correction of the variation in the fuel
injection start time will be described.

        Fig. 6 is a diagram for illustrating the correction
of the variation in the fuel injection start time. Part
(A) of Fig. 6 shows the relationship between the driving
20 signal and the fuel injection rate before the correction.
Part (B) of Fig. 6 shows the relationship between the
driving signal and the fuel injection rate after the
correction. In both Part (A) and Part (B) of Fig. 6, the
upper curve represents the driving signal and the lower
25 curve represents the fuel injection rate. In part (A) of
Fig. 6 also, the waveform X of the fuel injection rate at
the reference in-cylinder pressure is shown by the solid

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line and the waveform Y of the fuel injection rate at the
in-cylinder pressure of the engine is shown by the broken
line, in a similar manner as in Fig. 4. Reference signs b
and f in Part (A) of Fig. 6 correspond to reference signs

5   b and f in Part (A) of Fig. 4 respectively.

As will be seen from Part (A) of Fig. 6, in the case
of the reference in-cylinder pressure, there is a time
delay τd1 before the fuel injection is started after
generation of the fuel injection signal by the ECU 10. On

10   the other hand, in the case of the in-cylinder injection,
there is a time delay τd2 before the fuel injection is
started after generation of the fuel injection signal,
namely, the fuel injection is started earlier than in the
case of reference in-cylinder pressure by a time Δτd

15   (=τd1-τd2). In the following, time τd1 or τd2 will be
sometimes referred to as the fuel injection delay time τd.

It is already known that the time τd is
substantially proportional to the variation in the in-
cylinder pressure and the proportionality coefficient (i.e.

20   the sensitivity of the variation of the fuel injection
delay time τd) α varies depending on the rail pressure Pcr.
Their relationship is shown in Figs. 7 and 8. Fig. 7
shows the relationship between the proportionality
coefficient (or the sensitivity) α and the rail pressure

25   Pcr (specifically, the variation in the fuel injection
delay time per unit in-cylinder pressure relative to the
rail pressure). Fig. 8 shows the relationship between the

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variation in the fuel injection delay time τd (time Δτd)
and the variation in the in-cylinder pressure ΔPcl (=
Pcl(engine in-cylinder pressure) − Pcl'(reference in-
cylinder pressure)).

Thus, Δτd can be obtained from the following formula
(14) by calculating the rail pressure Pcr and the
variation in the in-cylinder pressure (Pcl − Pcl'). In
connection with this, it is preferable that the
relationship between the rail pressure Pcr and the
proportionality coefficient α be prepared as a map in
advance. Such a map constitutes the coefficient
calculation means 58.

$$\Delta\tau d = \alpha \times (Pcl - Pcl') \qquad --(14)$$

As shown in Part (B) of Fig. 6, it is possible to
correct a change in the fuel injection quantity caused by
a change in the fuel injection start time by shortening a
signal for opening the fuel injection valve 8 by time Δτd
obtained by formula (14). Furthermore, it is possible to
correct the fuel injection timing, which may be made
earlier by the influence of in-cylinder pressure, to a
desired fuel injection timing by delaying the fuel
injection timing by Δτd.

As described above, according to the present
invention, it is possible to calculate the variation in
the fuel injection quantity caused by a change in the fuel
injection rate and to calculate the variation in the fuel
injection start time. In the following, a process of

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controlling the power supply time over which electric
power is supplied to the fuel injection valve 8 in order
to open the fuel injection valve 8 will be described.

Fig. 9 is a flow chart for illustrating a process of
calculating a correction value for the fuel injection
quantity.

Firstly, in step S101, a required fuel injection
quantity is read by the ECU 10 in accordance with the
running state of the internal combustion engine 1. This
corresponds to calculation of a required fuel injection
quantity by the required fuel injection quantity
calculation means 55.

Next in step S102, the in-cylinder pressure (the in-
cylinder pressure of the engine) at the time when the fuel
injection is started is calculated. The calculation of
the in-cylinder pressure may be implemented by directly
detecting the in-cylinder pressure by detection means
provided for detecting the pressure in the cylinder 3 or
by estimating the in-cylinder pressure. The in-cylinder
pressure can be estimated based on, for example, the
pressure in the intake passage 4 and the ratio of the
interior volume of the cylinder (i.e. the ratio of the
volume during the fuel injection and the volume at the
bottom dead center). This corresponds to detection of the
in-cylinder pressure of the cylinder 3 by the in-cylinder
pressure detection means 61.

Next in step S103, the ratio of variation of the

fuel injection rate is calculated. The ratio of variation of the fuel injection rate is calculated by formula (11) presented before. This corresponds to calculation of the fuel injection rate by the fuel injection rate calculation means 56.

Next in step S104, the variation in the fuel injection quantity (or the correction value for the variation in the fuel injection rate) caused by a change in the fuel injection rate is calculated. This corresponds to calculation of the variation in the fuel injection quantity by the fuel injection quantity variation calculation means 57. This calculation is performed in accordance with the above-described method (calculation using formula (6) and formula (13)), and its result is taken into a step of determining a final designated fuel injection quantity that is executed in another routine. This step will be referred to as step S201 for facilitating description. In step S201, a final designated fuel injection quantity is calculated by correcting the required fuel injection quantity based on the variation in the fuel injection quantity calculated in step S104.

In step S105 succeeding step S104, the variation in the fuel injection start time is calculated by the start time variation calculation means 52. This calculation is executed in accordance with the above-described method, and its result is taken into a step of determining a final

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power supply time that is executed in another routine.
This step will be referred to as step S202 to facilitating
description.

In step S202, a power supply time is calculated
based on the required fuel injection quantity after
correction calculated in step S201 and the power supply
time is corrected based on the variation in the fuel
injection start time calculated in step S105. Thus, the
final power supply time is determined.

Here, it is preferable that the relationship between
the power supply time (i.e. the time period τ during
which the fuel injection valve 8 is opened) and the fuel
injection quantity (Q) be obtained in advance by
experiments and prepared as a τ-Q characteristic in the
form of a map. The power supply time can be calculated
based on the required fuel injection quantity after
correction using that τ-Q map. Then, it is possible to
determine the final power supply time by increasing or
decreasing the power supply time by a value corresponding
to the variation in the fuel injection start time. The
aforementioned map constitutes the fuel injection quantity
characteristic storing means 54.

Fuel injection with an accurate quantity can be
attained by applying the final power supply time thus
determined as power supply time to the fuel injection
valve 8. It should be understood that no limitation is
placed on the type of the above-described fuel injection

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valve 8 and this embodiment can be preferably applied to
an injection valve driven by a solenoid and an injection
valve driven by a piezoelectric element. In addition,
this embodiment can be preferably applied to a direct

5   drive injection valve having no control chamber by setting
the variation in the fuel injection delay time τd as 0 (no
variation).

<Embodiment 2>

10   In the second embodiment of the present invention,
in the case that execution of the fuel injection is
completed before the lift amount of the needle valve 23
reaches the full lift, the fuel injection quantity is
corrected in accordance with a method different from the

15 method described in connection with the first embodiment.
Fig. 10 is a block diagram showing the fuel injection
quantity variation calculation means 57 and related
portions in this embodiment. In this embodiment, the ECU
10 constitutes the suction chamber pressure calculation

20 means 57a and the unit fuel injection quantity variation
calculation means 57b.

        The fuel injection quantity variation calculation
means 57 is equipped with a suction chamber pressure
calculation means 57a for calculating pressure in a

25 suction chamber 26 formed in the tip end side of a valve
seat on/from which the needle valve 23 is to be
received/detached, based on the fuel pressure detected by

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the rail pressure sensor 11 and the position of said
needle valve 23; the unit fuel injection quantity
variation calculation means 57b for calculating a
variation in the fuel injection quantity per unit in-

5 cylinder pressure based on the fuel injection quantity
calculated by the required fuel injection quantity
calculation means 55 and the suction chamber pressure
calculated by the suction chamber pressure calculation
means57a.

10   When according to a result of the comparison by the
comparison means 60, the fuel injection quantity is less
than the estimated fuel injection quantity, the fuel
injection quantity variation calculation means 57
calculates the variation in the fuel injection quantity

15 based on a variation in the in-cylinder pressure detected
or estimated by the in-cylinder pressure detection means
61 relative to the reference in-cylinder pressure and the
variation in the fuel injection quantity per unit in-
cylinder pressure calculated by the unit fuel injection

20 quantity variation calculation means 57b.

        The basic structure of the internal combustion
engine of this embodiment is the same as that in the first
embodiment, and the parts in this embodiment same as those
in the first embodiment will be designated by the same

25 reference signs and descriptions thereof will be omitted.

        In the case that the needle valve 23 does not reach
the fully lifted state, especially in the case that fuel

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injection is effected in the state with a small lift

amount, the actual pressure in the fuel injection process

(the actual injection pressure) does not reach the rail

pressure. In view of this, in this embodiment, the fuel

5   injection quantity is corrected based on the pressure in

the suction chamber 26 or the so-called suction chamber

pressure (namely, the pressure actually applied to the

injection hole (equivalent to the actual injection

pressure)).

10   The suction chamber pressure is determined by the

lift position of the needle valve 23 and the actual rail

pressure detected by the rail pressure sensor 11. The

lift position of the needle valve 23 is determined by the

time and the lifting speed of the needle valve 23. The

15   lifting speed of the needle valve 23 can be determined

based on the characteristics of the orifice flow flowing

out of the control camber 32 through the outlet orifice

34a. The movement can be assumed to be uniform motion,

that is the lifting speed of the needle valve 23 can be

20   assume to be substantially constant. Therefore, the

suction chamber pressure can be represented as a function

of the time, and it can be obtained based on the time.

This corresponds to calculation of the pressure in the

suction chamber 26 by the suction chamber pressure

25   calculation means 57a.

Thus, if a map for obtaining the variation in fuel

injection quantity per unit in-cylinder pressure based on

the required fuel injection quantity calculated by the
required fuel injection quantity calculation means 55 and
the suction chamber pressure is prepared in advance by,
for example, experiments, it is possible to calculate the
5   variation in the fuel injection quantity based on the
variation in fuel injection quantity per unit in-cylinder
pressure and the variation in the in-cylinder pressure of
the engine relative to the reference in-cylinder pressure.
The aforementioned map constitutes the unit fuel injection
10   quantity variation calculation means 57b.

According to this embodiment, since in the case that
the fuel injection is completed before the lift amount of
the needle valve 23 reaches the full lift, a correction
value is calculated based on the suction chamber pressure
15   that is equivalent to the actual injection pressure,
correction with improved accuracy can be realized. In
connection with this process, determination as to whether
the execution of the fuel injection is completed before
the lift amount of the needle valve 23 reaches the full
20   lift or not is determined based on the result of the
comparison effected by the comparison means 60.

<Embodiment 3>

In the third embodiment of the present invention,
25   correction of the fuel injection quantity is performed
based on the rail pressure Pcr and the in-cylinder
pressure of the engine Pcl, unlike with the first

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embodiment. Specifically, when the in-cylinder pressure
of the engine Pcl increases, it is assumed that the rail
pressure Pcr has decreased, and fuel injection time is
calculated while compensating the variation in the fuel

5   injection quantity due to a change in the fuel injection
rate based on the difference between the rail pressure Pcr
and the in-cylinder pressure of the engine Pcl. In
addition, when the in-cylinder pressure of the engine Pcl
increases, it is assumed that the rail pressure Pcr has

10   increased, and variation in fuel injection start time is
calculated based on the sum of the rail pressure Pcr and
the in-cylinder pressure of the engine Pcl.

     Fig. 11 is a block diagram of the fuel injection
control apparatus in this embodiment.

15   As shown in Fig.11, the fuel injection control
apparatus according to this embodiment is equipped with
the time period calculation means 51, start time variation
calculation means 52 and control means 53.

     The time period calculation means 51 calculates the

20   fuel injection time period being corrected to compensate a
variation in the fuel injection quantity caused by a
variation in the fuel injection rate due to a variation in
the in-cylinder pressure detected or estimated by the in-
cylinder pressure detection means 61 relative to a

25   reference in-cylinder pressure that is stored in advance.

     The start time variation calculation means 52
calculates a variation in the fuel injection start time at

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the in-cylinder pressure detected or estimated by in-cylinder pressure detection means 61 relative to the fuel injection start time at the reference in-cylinder pressure described later.

The control means 53 controls the time period over which fuel is injected from the fuel injection valve 8 based on the fuel injection time period calculated by the time period calculation means 51 and the variation in the fuel injection start time calculated by the start time variation calculation means 52.

The fuel injection control apparatus according to this embodiment is further equipped with the rail pressure sensor 11 for detecting the pressure of the high pressure fuel supplied to the fuel injection valve 8;the fuel injection quantity characteristic storing means 54 for storing a characteristic, in relation to valve opening time of the fuel injection valve 8, of the fuel injection quantity injected by the fuel injection valve 8 during the valve opening time in accordance with the pressure of the high pressure fuel supplied to the fuel injection valve 8; the required fuel injection quantity calculation means 55 for calculating a desired fuel injection quantity based on the running state of the internal combustion engine; the first virtual fuel pressure calculation means 65 for calculating a first virtual fuel pressure by subtracting a variation in the in-cylinder pressure detected or estimated by the in-cylinder pressure detection means 61

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relative to the reference in-cylinder pressure from the
fuel pressure detected by the rail pressure sensor 11; the
second virtual fuel pressure calculation means 66 for
calculating a second virtual fuel pressure by adding a
5 variation in the in-cylinder pressure detected or
estimated by the in-cylinder pressure detection means 61
relative to the reference in-cylinder pressure to the fuel
pressure detected by the rail pressure sensor 11; the
injection delay time calculation means 67 for calculating
10 injection delay time from the time at which a signal for
opening the fuel injection valve 8 is generated to the
time at which fuel injection by the fuel injection valve 8
is started, based on the fuel pressure detected by the
rail pressure sensor 11.

15   The time period calculation means 51 calculates the
fuel injection time period utilizing the fuel injection
quantity characteristic storing means 54 based on the
first virtual fuel pressure calculated by the first
virtual fuel pressure calculation means 65 and the fuel
20 injection quantity calculated by the required fuel
injection quantity calculation means 55.

     The start time variation calculation means 52
calculates, by means of the fuel injection delay time
calculation means 67, a fuel injection delay time for the
25 fuel pressure detected by the rail pressure sensor 11 and
a fuel injection delay time for the second virtual fuel
pressure calculated by the second virtual fuel pressure

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calculation means 66 and calculates the variation in the
fuel injection start time from a difference between those
injection delay times.

In this embodiment, the ECU 10 constitutes the first
virtual fuel pressure calculation means 65, the second
virtual fuel pressure calculation means 66 and the
injection delay time calculation means 67. The basic
structure of the internal combustion engine 1 of this
embodiment is the same as that in the first embodiment,
and the parts in this embodiment same as those in the
first embodiment will be designated by the same reference
signs and descriptions thereof will be omitted.

Fig. 12 shows a τ-Q map serving as the fuel
injection quantity characteristic storing means 54 in this
embodiment.

In Fig. 12, the solid curve indicates the τ-Q
characteristic at the rail pressure Pcr(i.e. at the
reference in-cylinder pressure Pcl'). When the required
fuel injection quantity is Q, the power supply time for
attaining the required fuel injection quantity Q at the
rail pressure Pcr(at the reference in-cylinder pressure
Pcl') determined from the τ-Q characteristic indicated by
the solid curve in Fig. 12 is τl.

In Fig. 12, the broken curve indicates the τ-Q
characteristic at the first virtual rail pressure (Pcr-
ΔPcl) obtained by subtracting the difference ΔPcl of the
in-cylinder pressure of the engine Pcl and the reference

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in-cylinder pressure Pcl' from the rail pressure Pcr under
the condition in which the in-cylinder pressure of the
engine Pcl is applied to the rail pressure Pcr. The power
supply time for attaining the required fuel injection

5   quantity Q at the first virtual rail pressure (Pcr-ΔPcl)
determined from the τ-Q characteristic indicated by the
broken curve in Fig. 12 is τ2.

As per the above, since in the stationary running
state, the flow rate through the injection hole can be

10   determined based on the pressure difference between the
interior and the exterior, injection with a required fuel
injection quantity can be attained by replacing the rail
pressure with the pressure difference between the rail
pressure and the in-cylinder pressure of the engine as the

15   virtual rail pressure directly. Therefore, in the
stationary running state, additional calculation for
correction is not required and errors through calculation
hardly occur. The use of the pressure difference between
the rail pressure and the in-cylinder pressure of the

20   engine as a virtual rail pressure corresponds to
calculation of the first virtual fuel pressure by the
first virtual fuel pressure calculation means 65.

However, the variation in the fuel injection start
time is not corrected even when the pressure difference is

25   substituted for the rail pressure. This is because at the
time of opening the needle valve 23, the area on which the
in-cylinder pressure acts is small as compared to the area

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on which the fuel pressure acts, and because the deceasing speed of the fluid pressure in the control chamber 32 varies depending on the rail pressure.

In view of this, in this embodiment, correction of the fuel injection start time is effected in the following manner.

Fig. 13 shows the relationship between the rail pressure Pcr and the fuel injection delay time τd in this embodiment. The map containing this relationship constitutes the fuel injection delay time calculation means 67.

From the relationship shown in Fig. 13, the time difference Δτd between the fuel injection delay time at the rail pressure Pcr and the fuel injection delay time at a second virtual rail pressure (Pcr+ΔPcl) obtained by adding the difference ΔPcl of the in-cylinder pressure of the engine Pcl and the reference in-cylinder pressure Pcl' to the rail pressure Pcr is obtained.

A τ-Q characteristic in which the difference Δτd between the fuel injection delay time at the rail pressure Pcr and the fuel injection delay time at the second virtual rail pressure (Pcr+ΔPcl) has been compensated is indicated as the dashed line in Fig. 12. In this case, the power supply time for attaining the required fuel injection quantity Q is τ3. The power supply time τ3 can be represented by the following formula (15).

$$\tau 3 = \tau 2 - \Delta \tau d \qquad --(15)$$

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When the final power supply time thus determined is applied as the power supply time for the fuel injection valve 8, the fuel injection can be effected with improved accuracy in the injection quantity. The above-described process of obtaining the second virtual rail pressure (Pcr+ΔPcl) by adding the difference ΔPcl of the in-cylinder pressure of the engine Pcl and the reference in-cylinder pressure Pcl' to the rail pressure Pcr corresponds to calculation of the second virtual fuel pressure by the second virtual pressure calculation means 66.

Industrial Applicability

According to the present invention, it is possible to control the fuel injection quantity that changes with changes in the in-cylinder pressure with improved accuracy, and it is possible to attain a target fuel injection quantity irrespectively of the running state of the internal combustion engine.